

## **Temperature of Combustion Gases from Natural Gas**

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### Abstract

Combustion gases from natural gas are mixture of five individual gases. They origin is the combustion of natural gas, ten individual gases with atmospheric air, five individual gases. Authors present in their works thermodynamic parameters of this combustion gas with help of thermodynamics of gas mixtures. In this paper they present their method of determination of temperature of hot combustion gases from natural gas 2010, and one concrete example with numerical results.

Key words: Natural gas, Combustion gases, Temperature of gas mixtures

### Introduction

Combustion gases from natural gas are mixture of five individual gases, nitrogen, oxygen, carbon dioxid, argon and water vapour. Their source is combustion of natural gas, which is mixture of ten hydrocarbons and of atmospheric air, which is mixture of five individual gases, as source of oxygen. All this gases have parameters in dependence of temperature in different exponential dependences. Authors used in their work exact values of its thermodynamic parameters from thermal tables in (RAZNJEVIČ, 1969).

In professional literature is thermodynamic of combustion gas nearly only analysed with thermodynamics of one ideal gas. Calculations are therefore not exact.

Authors elaborated therefore their method for exact parameters of combustion gases of gas fuels with thermodynamic of gas mixtures and with using of exact values of thermodynamic parameters, as in their work (VITÁZEK-HAVELKA, 2008).

In this paper they present one part of their method for destination of exact temperature of hot combustion gas

### Material and methods

Authors appreciate in this paper combustion gas of natural gas which was delivered by SPP a.s. in Bratislava in the year 2010. SPP presents every months in Internet (<u>www.spp.sk</u>, 2010) composition of natural gas, delivered in Slovakia and some basic properties from laboratory specimens.

Authors used in their work average values of these statements calculated for the whole year 2010 presented in their work (VITÁZEK-HAVELKA, 2011).

Authors used exact values of thermodynamic parameters of individual gases from tables in (RAŽNJEVIČ, 1969), where are tables of enthalpy of gases for temperatures from 0°C to 3000°C. They divided this temperature span in four sections:

 $0 \div 100^{\circ}C$  for climatisation and drying of seed materials,

 $100 \div 500^{\circ}$ C for drying of cereal grains and other agricultural products for consum,

 $500 \div 1000^{\circ}C$  for drying of food materials, lucerne, hay and sim..,

 $1000 \div 2400^{\circ}$ C for boilers and heat exchangers.

Authors derived in every section with linear regression linear relations for enthalpy:



Authors used in this paper their stechiometric parameters of natural gas 2010 from their work (VITÁZEK-HAVELKA, 2011).

Authors analyse in their work an indefinite establishment with thermal efficiency of burning  $\eta_h$  and with air surplus coefficient  $\alpha$ .

Authors consider combustion gases as mixture of stechiometric combustion gases and of air surplus  $m_p$  with surplus coefficient  $\alpha$ :

$$a = \frac{m_{iv} + m_p}{m_{iv}} \tag{1}$$

Authors describe mass quantities relations of individual gases for mass of natural gas  $m_{ng} = 1$  kg. Mass quantities of individual gases in combustion gas are described with relations:

Nitrogen N<sub>2</sub>  $m_{N2} = m_{iv} \cdot \alpha \cdot \sigma_{aN2} + \sigma_{ngN2}$  (2)

Oxygen O<sub>2</sub> 
$$m_{O2} = m_{iv} \cdot (\alpha - 1) \cdot \sigma_{aO2}$$
 (3)

Carbon dioxide CO<sub>2</sub>  $m_{CO2} = m_{iv} \cdot \alpha \cdot \sigma_{aCO2} + \Delta m_{CO2} + \sigma_{ngCO2}$  (4)

Argon et high. Ar 
$$m_{Ar} = m_{iv} \cdot \alpha \cdot \sigma_{aAr}$$
 (5)

Water vapour H<sub>2</sub>O 
$$m_w = m_{iv} \cdot \alpha \cdot x_o + \Delta m_w$$
 (6)

Whole mass of combustion gases from 1 kg of natural gas is:

$$n_{cs} = \sum m_{is} \tag{7}$$

Enthalpy of combustion gases calculated from basic temperature 0°C, without any latent heat, is:

$$i_{s} = \sum \boldsymbol{S}_{si} \cdot i_{si} = \sum \frac{m_{si} \cdot i_{si}}{m_{cs}}$$
 kJ/kg (8)

The function i = f(t) is not linear, but exponential. Authors therefore divided the whole temperature area in mentioned temperature sections. They derived in every section with linear regression this linear relation:

$$i = a + b.t$$
 kJ/kg (9)

In all sections the value of coefficient of regression is better as 0.999.

Enthalpy of combustion gases calculated with this table values, is:

$$i_{st} = \sum s_{si} . (a_{si} + b_{si} . t_s) = \sum \frac{m_{si} . (a_{si} + b_{si} . t_s)}{m_{cs}} \qquad \text{kJ/kg}$$
(10)

Enthalpy of combustion gases, calculated with heat value  $q_n$  of natural gas is sum of increase of enthalpy  $\Delta i_1$  of combustion gases between ambient temperature  $t_o$  and temperature of hot combustion gases  $t_s$ , and of complement of enthalpy  $\Delta i_o$  for temperature section from  $t_o$  to 0°C:

$$i_{st} = \Delta i_1 + \Delta i_0 = \frac{q_n \cdot h_h}{m_{cs}} + \sum \frac{m_{si} \cdot c_{pi} \cdot t_0}{m_{cs}} \qquad \text{kJ/kg}$$
(11)

Authors derived following relation for calculation of the temperature of hot combustion gases:

$$i_{sq} = \sum \frac{m_{si} \cdot (a_{si} + b_{si} \cdot t_s)}{m_{cs}}$$
 kJ/kg (12)

From this eq. they derived final relation for temperature of hot combustion gases:

$$t_s = \frac{i_s \cdot m_{cs} - \sum m_{si} \cdot a_{si}}{\sum m_{si} \cdot b_{si}}$$
 °C (13)



Physics – Research - Applications – Education 2011

# **Results and Discussion**

Example						
Given values:	1500	100 l-D-			<u>(00)</u>	
Atmospheric air: temperature $t_o$ Mass concentra	= 15°C, pressure $p_a$ tions of individual g	ases selecte	d from (C	$\phi$ CHYSKÝ,	= 60% 1977)	
Specific moist	ure, from i-x diagram	m of wet air	$x_o = 6.5$	g/kg		
Establishment: Thermal efficie Air surplus coef	blishment: Thermal efficiency of the burner $\eta_h = 98\%$ Air surplus coefficient $\alpha = 1.1$					
Natural gas 2010, from (VITAZ Stechiometric mass of dry air fo With combustion of 1kg of natu Carbon dioxide	EK-HAVELKA, 20 or 1 kg of natural gas ral gas we obtain ne $\Delta m_{CO2} = 2$	011) $m_{iv} = 16.89$ $m_{iv} = 16.89$ $m_{iv} = 16.89$ $m_{iv} = 16.89$ $m_{iv} = 16.89$ $m_{iv} = 16.89$ $m_{iv} = 16.89$	9 kg			
Water vapour	$\Delta m_w = 2$	2.1866 kg				
Heat value of natural ga	as 2010 $q_n = 4$	48 883 kJ/kg	<b>r</b>			
From 1 kg of burned natural gas Carbon dioxide	s remain inert gases: $\sigma_{ngCO2} =$	0.0047 kg/k	g			
Nitrogen	$\sigma_{ngN2} =$	0.0139 kg/k	g			
Temperature of combustion gas	es will be in the area	a 1000÷2400	с°С.			
Authors' therefore used relat	ions of linear reg	ression of	enthalpy	in the t	emperature	section
1000÷2400°C: Nitrogen	$i_{\rm V2} = 1.2778 \ t = 170$	) 32	kI/ko	r = 0.999	75 (14)	
Ovugon	$i_{N2} = 1.2776.t$ 170	73	kJ/kg	r = 0.000	(14)	
Oxygen Corbon dioxido	$i_{02} = 1.178.t - 138$	15 7	NJ/Ng	n = 0.0000	(13)	
	$i_{CO2} = 1.3395.t - 24$	+3.7	KJ/Kg	r = 0.9999	95 (10)	
Argon + hign.	$i_{Ar} = 0.523.t$				(17)	
Water vapour	$i_w = 2.2574.t - 733$	5.09	kJ/kg	r = 0.999	63 (18)	
Calculated mass quantities of in Nitrogen	dividual gases for co $m_{N2} = 14.069$	ombustion o	f 1kg nat kg/kg	ural gas:		
Oxygen	$m_{O2} = 0.3909$		kg/kg			
Carbon dioxide	$m_{CO2} = 2.9511$		kg/kg			
Argon +	$m_{Ar} = 0.0093$		kg/kg			
Water vapour	$m_w = 2.2574$		kg/kg			
Whole mass	$m_{cs} = 19.6777$		kg/kg			
Introduction in the relation for a	combustion gas enth	alnv	00			
$i_{re} = \Delta i_1 + \Delta i_0 = 2.434 + 16.6 = 2.451.6$			kJ/kg		(19)	
$\sum_{m=1}^{M} m_{\perp} a_{\perp} = -4.840$				U		(20)
= 28.81						(21)
And final relation for temperatu	re of combustion ga	ses:				
$t_s = \frac{2451.0117.07144040}{28.81} = 1841.9$				°C		(22)



Presented method describes only one thermodynamic process, process of burning of natural gas, in which mixture of natural gas and atmospheric air is transformed in mixture of combustion gases with effective output of heat.

Authors are not studying the effectivity of this process of burning. This phenomenon they express only with one coefficient of thermal effectivity of the burner  $\eta_h$ .

As a result of using exact thermodynamic parameters from thermal tables is the possibility to calculate exact values of enthalpy and temperature of combustion gases. This method is above first established for modelling of combustion of natural gas in establishment with prescribed conditions.

Presented method is time consuming. Therefore for greater amount of calculations it is advantageous to use a computer with pertinent computer program.

### Conclusion

Presented method enables to calculate exact values of temperature and other thermodynamic parameters of combustion gases from natural gas.

Authors use this presented system as part of a computer program in Q-Basic for calculation of all desired thermodynamic parameters of combustion gases from gas fuels.

Authors present this paper as example of research and development work in their school activity.

### Acknowledgement

This paper is published thanks to the grant project VEGA 1/0708/09. "Research into the use of thermal energy from renewable resources in agricultural drying with respect to its impact on ecology and technology."

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